

Насос аксиально-поршневой A10VSO18 (31 серии)



- ▶ All-purpose medium pressure pump
- ▶ Sizes
- ▶ Nominal pressure 280 bar
- ▶ Maximum pressure 350 bar
- ▶ Open circuit

Features

- ▶ Variable pump with axial piston rotary group in swashplate design for hydrostatic drives in open circuit.
- ▶ Flow is proportional to drive speed and displacement.
- ▶ The flow can be infinitely varied by adjusting the swashplate.
- ▶ 2 drain ports
- ▶ Excellent suction characteristics
- ▶ Low noise level
- ▶ Long service life
- ▶ Good power to weight ratio
- ▶ Versatile controller range
- ▶ Short control time
- ▶ The through drive is suitable for adding gear pumps and axial piston pumps up to the same size, i.e., 100% through drive.
- ▶ Suitable for operation with mineral oil and HF hydraulic fluids

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Type code

01	02	03	04	05	06	07	08	09	10	11	12	13
	A10VS	O			/	31		-				

Version		18	28	45	71	88	100
01	Standard version for mineral oil (without code)	•	•	•	•	•	•
	HFA, HFB, HFC hydraulic fluid	•	•	•	•	•	E
	High-speed version (external dimensions are not affected by this option).	-	-	•	•	-	H

Axial piston unit

02	Swashplate design, variable, nominal pressure 280 bar, maximum pressure 350 bar	•	•	•	•	•	A10VS
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Operating mode

03	Pump, open circuit						O
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Size (NG)

04	Geometric displacement, see table of values on page 8 and 9	18	28	45	71	88	100
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Control device

		18	28	45	71	88	100
05	Two-point control, direct operated	•	•	•	•	•	DG
	Pressure controller hydraulic	•	•	•	•	•	DR
	with flow controller hydraulic X-T open	•	•	•	•	•	DFR
	X-T plugged, with flushing function	•	•	•	•	•	DFR1
	with pressure cut-off hydraulic remote controlled	•	•	•	•	•	DRG
	electric negative control U = 24 V	•	•	•	•	•	ED72
	electric positive control U = 24 V	•	•	•	•	•	ER72
	Pressure, flow and power controller	-	•	•	•	•	DFLR

Series

06	Series 3, index 1						31
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Direction of rotation

		18	28	45	71	88	100
07	Viewed on drive shaft clockwise	•	•	•	•	•	R
	counter-clockwise	•	•	•	•	•	L

Sealing material

		18	28	45	71	88	100
08	FKM (fluorocarbon rubber)	•	•	•	•	•	V
	NBR (nitrile rubber) only with use of HFA, HFB, HFC hydraulic fluid (position 01; order code "E")	•	•	•	•	•	P

Drive shaft

		18	28	45	71	88	100
09	Splined shaft Standard shaft	•	•	•	•	•	S
	ISO 3019-1 similar to shaft "S" however for higher torque	•	•	•	•	-	R
	Parallel keyed shaft permissible through-drive torque (see page 10)	•	•	•	•	•	P
	DIN 6885						

01	02	03	04	05	06	07	08	09	10	11	12	13
	A10VS	O			/	31		-				

Mounting flange

Mounting flange				18	28	45	71	88	100	
10	ISO 3019-2	2-hole		●	●	●	●	●	●	A

Working port

Working port			18	28	45	71	88	100	
11	SAE flange connections according to ISO 6162 fastening thread, metric	laterally opposite	●	●	●	–	–	●	12
			–	–	–	●	●	–	42

Through drive (for mounting options, see page 41)

12	For flange ISO 3019-1	Hub for splined shaft ¹⁾							
	Diameter	Diameter							
	without through drive		18	28	45	71	88	100	N00
	82-2 (A)	5/8 in 9T 16/32DP	●	●	●	●	●	●	K01
		3/4 in 11T 16/32DP	●	●	●	●	●	●	K52
	101-2 (B)	7/8 in 13T 16/32DP	-	●	●	●	●	●	K68
		1 in 15T 16/32DP	-	-	●	●	●	●	K04
	127-2 (C)	1 1/4 in 14T 12/24DP	-	-	-	●	●	●	K07
		1 1/2 in 17T 12/24DP	-	-	-	-	-	●	K24
	For flange ISO 3019-2								
	Diameter		18	28	45	71	88	100	
	80, 2-hole	3/4 in 11T 16/32DP	●	●	●	●	●	●	KB2
	100, 2-hole	7/8 in 13T 16/32DP	-	●	●	●	●	●	KB3
		1 in 15T 16/32DP	-	-	●	●	●	●	KB4
	125, 2-hole	1 1/4 in 14T 12/24DP	-	-	-	●	●	●	KB5
		1 1/2 in 17T 12/24DP	-	-	-	-	-	●	KB6
	Ø63, metric 4-hole	Shaft key Ø25	-	●	●	●	●	●	K57

Connector for solenoids²⁾

Connector for solenoids ²⁾		18	28	45	71	88	100	
13	Without connector (without solenoid, only for hydraulic controls, without code)	●	●	●	●	●	●	
	HIRSCHMANN connector – without suppressor diode	●	●	●	●	●	●	H

● = Available ○ = On request - = Not available

Notice

- Observe the project planning notes on page 47 and the project planning notes regarding each control device.
- In addition to the type code, please specify the relevant technical data when placing your order.

¹⁾ Hub for splined shaft according to ANSI B92.1a
(drive shaft allocation according to ISO 3019-1)

²⁾ Connectors for other electric components can deviate.

Hydraulic fluids

The A10VSO variable pump is designed for operation with HLP mineral oil according to DIN 51524-2.

See the following data sheets for application instructions and requirements for hydraulic fluids before the start of project planning:

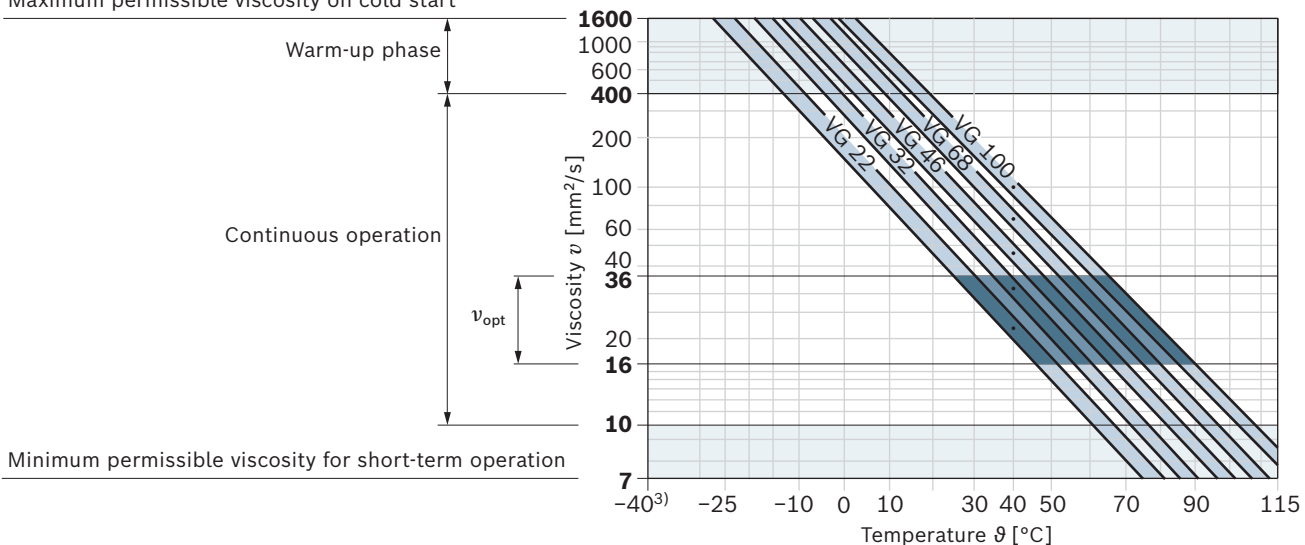
- ▶ 90220: Hydraulic fluids based on mineral oils and related hydrocarbons
- ▶ 90221: Environmentally acceptable hydraulic fluids
- ▶ 90222: Fire-resistant, water-free hydraulic fluids (HFDR/HFDU) (for permissible technical data, see data sheet 90225)
- ▶ 90223: Fire-resistant, water-containing hydraulic fluids (HFAE, HFAS, HFB, HFC)
- ▶ 90225: Limited technical data for operation with water-free and water-containing fire-resistant hydraulic fluids (HFDR, HFDU, HFB, HFC) – technical data

Viscosity and temperature of hydraulic fluids

	Viscosity	Shaft seal	Temperature ²⁾	Remarks
Cold start	$v_{\max} \leq 1600 \text{ mm}^2/\text{s}$	FKM	$\vartheta_{\text{st}} \geq -25 \text{ }^{\circ}\text{C}$	$t \leq 3 \text{ min}$, without load ($p \leq 50 \text{ bar}$), $n \leq 1000 \text{ rpm}$ Permissible temperature difference between axial piston unit and hydraulic fluid in the system maximum 25 K
Warm-up phase	$v = 1600 \dots 400 \text{ mm}^2/\text{s}$			$t \leq 15 \text{ min}$, $p \leq 0.7 \times p_{\text{nom}}$ and $n \leq 0.5 \times n_{\text{nom}}$
Continuous operation	$v = 400 \dots 10 \text{ mm}^2/\text{s}^{1)}$	FKM	$\vartheta \leq +110^{\circ}\text{C}$	Measured at port L , L₁
	$v_{\text{opt}} = 36 \dots 16 \text{ mm}^2/\text{s}$			Optimal operating viscosity and efficiency range
Short-term operation	$v_{\min} = 10 \dots 7 \text{ mm}^2/\text{s}$	FKM	$\vartheta \leq +110^{\circ}\text{C}$	$t \leq 3 \text{ min}$, $p \leq 0.3 \times p_{\text{nom}}$, measured at port L , L₁

▼ Selection diagram

Maximum permissible viscosity on cold start



1) This corresponds, for example on the VG 46, to a temperature range of +4 °C to +85 °C (see selection diagram)

2) If the temperature at extreme operating parameters cannot be adhered to, please contact us.

3) For applications in the low-temperature range, please contact us.

Filtration of the hydraulic fluid

Finer filtration improves the cleanliness level of the hydraulic fluid, which increases the service life of the axial piston unit.

A cleanliness level of at least 20/18/15 is to be maintained according to ISO 4406

At a hydraulic fluid viscosity of less than 10 mm²/s (e.g. due to high temperatures during short-term operation), at the drain port, a cleanliness level of at least 19/17/14 under ISO 4406 is required.

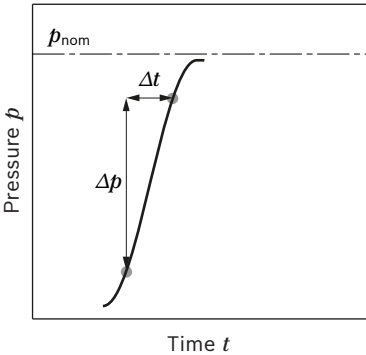
For example, viscosity corresponds to 10 mm²/s at:

- HLP 32 a temperature of 73 °C
- HLP 46 a temperature of 85 °C

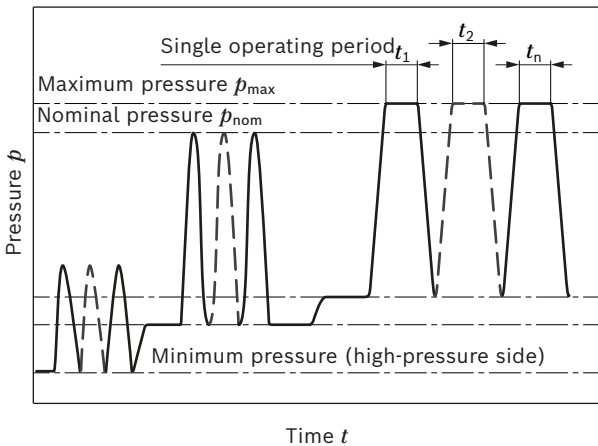
Working pressure range

Pressure at working port B			Definition
Nominal pressure p_{nom}	280 bar	The nominal pressure corresponds to the maximum design pressure. The maximum pressure corresponds to the maximum working pressure within a single operating period. The sum of single operating periods must not exceed the total operating period.	
Maximum pressure p_{max}	350 bar		
Single operating period	2 ms		
Total operating period	300 h		
Minimum pressure $p_{B absolute}$ (high-pressure side)	10 bar ¹⁾	Minimum pressure on the high-pressure side (B) which is required in order to prevent damage to the axial piston unit.	
Rate of pressure change $R_{A max}$	16000 bar/s	Maximum permissible pressure build-up and reduction speed during a pressure change across the entire pressure range.	
Pressure at suction port S (inlet)			
Minimum pressure $p_{S min}$	Standard	0.8 bar absolute	Minimum pressure at suction port S (inlet) which is required to prevent damage to the axial piston unit. The minimum pressure depends on the rotational speed and displacement of the axial piston unit.
Maximum pressure $p_{S max}$		10 bar	
Case pressure at port L, L ₁			
Maximum pressure $p_{L max}$		2 bar ¹⁾ absolute	Maximum 0.5 bar higher than inlet pressure at port S , but not higher than $p_{L max}$. A drain line to the reservoir is required.
Pilot pressure port X with external high pressure			
Maximum pressure p_{max}	350 bar	When designing all control lines with external high pressure, the values for the rate of pressure change, maximum single operating period and total operating period applicable to port B must not be exceeded.	

▼ Rate of pressure change $R_{A max}$



▼ Pressure definition



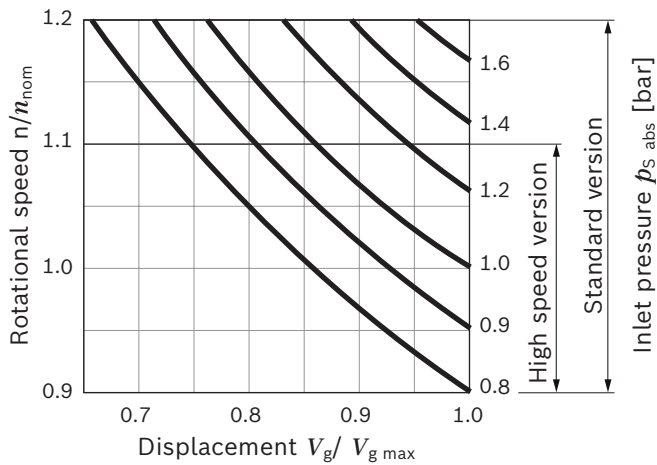
Total operating period = $t_1 + t_2 + \dots + t_n$

- Notice
- ▶ Working pressure range applies when using hydraulic fluids based on mineral oils. Please contact us for values for other hydraulic fluids.
 - ▶ In addition to the hydraulic fluid and the temperature, the service life of the shaft seal is influenced by the rotational speed of the axial piston unit and the case pressure.
 - ▶ The case pressure must be greater than the external pressure (ambient pressure) at the shaft seal.

1) Other values on request

Minimum permissible inlet pressure at suction port **S** with speed increase

In order to avoid damage to the pump (cavitation), a minimum inlet pressure must be guaranteed at suction port **S**. The minimum inlet pressure level depends on the rotational speed and the displacement of the variable pump.



During continuous operation in overspeed over n_{nom} , a reduction in operational service life is to be expected due to cavitation erosion.

Technical data, standard unit

Size			18	28	45	71	88	100
Geometric displacement, per revolution			18	28	45	71	88	100
Maximum rotational speed ¹⁾	at $V_{g \max}$	n_{nom} rpm	3300	3000	2600	2200	2100	2000
	at $V_g < V_{g \max}^{2)}$	$n_{\text{max perm}}$ rpm	3900	3600	3100	2600	2500	2400
Flow	at n_{nom} and $V_{g \max}$	$q_{v \max}$ l/min	59	84	117	156	185	200
	at $n_E = 1500$ rpm and $V_{g \max}$	$q_{vE \max}$ l/min	27	42	68	107	132	150
Power at $\Delta p = 280$ bar	with n_{nom} , $V_{g \max}$	$P_{\max}$ kW	28	39	55	73	86	93
	at $n_E = 1500$ rpm and $V_{g \max}$	$P_{E \max}$ kW	12.6	20	32	50	62	70
Torque with $V_{g \max}$ and	$\Delta p = 280$ bar	$M_{\max}$ Nm	80	125	200	316	392	445
	$\Delta p = 100$ bar	M Nm	30	45	72	113	140	159
Rotary stiffness of drive shaft	S	c Nm/rad	11087	22317	37500	71884	71884	121142
	R	c Nm/rad	14850	26360	41025	76545	76545	–
	P	c Nm/rad	13158	25656	41232	80627	80627	132335
Moment of inertia of the rotary group			J_{TW} kgm ²	0.00093	0.0017	0.0033	0.0083	0.0083
Case volume			V l	0.4	0.7	1.0	1.6	1.6
Weight without through drive (approx.)			m kg	12.9	18	23.5	35.2	35.2
Weight with through drive (approx.)				14	19.3	25.1	38	38

Determination of the characteristics		
Flow	$q_v = \frac{V_g \times n \times \eta_v}{1000}$	[l/min]
Torque	$M = \frac{V_g \times \Delta p}{20 \times \pi \times \eta_{mh}}$	[Nm]
Power	$P = \frac{2 \pi \times M \times n}{60000} = \frac{q_v \times \Delta p}{600 \times \eta_t}$	[kW]
Key		
V_g Displacement per revolution [cm ³]		
Δp Differential pressure [bar]		
n Rotational speed [rpm]		
η_v Volumetric efficiency		
η_{hm} Hydraulic-mechanical efficiency		
η_t Total efficiency ($\eta_t = \eta_v \times \eta_{hm}$)		

Notice

- Theoretical values, without efficiency and tolerances; values rounded
- Operation above the maximum values or below the minimum values may result in a loss of function, a reduced service life or in the destruction of the axial piston unit. Bosch Rexroth recommends checking the loading by means of test or calculation / simulation and comparison with the permissible values.

- 1) The values are applicable:
- for the optimum viscosity range from $\nu_{\text{opt}} = 36$ to $16 \text{ mm}^2/\text{s}$
 - with hydraulic fluid based on mineral oils
 - At an abs. pressure $p_{\text{abs}} = 1 \text{ bar}$ at the suction port **S**
- 2) For a speed increase up to $n_{\text{max perm}}$, please observe the diagram on page 7.

Technical data, high-speed version

(external dimensions correspond to the standard unit)

Size		NG		45	71	100
Geometric displacement, per revolution		$V_{g \max}$	cm ³	45	71	100
Rotational speed maximum ¹⁾	at $V_{g \max}$	n_{nom}	rpm	3000	2550	2300
	at $V_g < V_{g \max}$ ²⁾	$n_{\text{max perm}}$	rpm	3300	2800	2500
Flow	at n_{nom} and $V_{g \max}$	$q_{v \max}$	l/min	135	178	230
Power	at n_{nom} , $V_{g \max}$ and $\Delta p = 280$ bar	P_{max}	kW	63	83	107
Torque	$\Delta p = 280$ bar	M_{max}	Nm	200	316	445
with $V_{g \max}$ and	$\Delta p = 100$ bar	M	Nm	72	113	159
Rotary stiffness	S	c	Nm/rad	37500	71884	121142
Drive shaft	R	c	Nm/rad	41025	76545	–
	P	c	Nm/rad	41232	80627	132335
Moment of inertia of the rotary group		J_{TW}	kgm ²	0.0033	0.0083	0.0167
Case volume		V	l	1.0	1.6	2.2
Weight without through drive (approx.)		m	kg	23.5	35.2	49.5
Weight with through drive (approx.)				25.1	38	55.4

Notice

- Theoretical values, without efficiency and tolerances; values rounded
- Operation above the maximum values or below the minimum values may result in a loss of function, a reduced service life or in the destruction of the axial piston unit. Bosch Rexroth recommends checking the loading by means of test or calculation / simulation and comparison with the permissible values.

Technical data, HF hydraulic fluids

Maximum rotational speed

Maximum rotational speed											
Hydraulic fluid ³⁾ E-version	Size		NG			18	28	45	71	88	100
HFA	at nominal pressure p_N	140 bar	n_{nom}	rpm		2450	2250	1950	1650	1550	1500
	at maximum pressure p_{max}	160 bar									
HFB	at nominal pressure p_N	140 bar	n_{nom}	rpm		2650	2400	2100	1760	1650	1600
	at maximum pressure p_{max}	160 bar									
HFC	at nominal pressure p_N	175 bar	n_{nom}	rpm		2650	2400	2100	1760	1650	1600
	at maximum pressure p_{max}	210 bar									
Technical data, HFD hydraulic fluids											
HFDR, HFDU polyalkylene glycol	at nominal pressure p_N	280 bar	n_{nom}	rpm		2650	2400	2100	1760	1650	1600
HFDU polyol ester	at nominal pressure p_N	280 bar				3300	3000	2600	2200	2100	2000

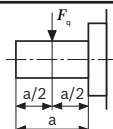
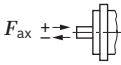
1) The values are applicable:

- At an abs. pressure $p_{\text{abs}} = 1$ bar at the suction port **S**
- for the optimum viscosity range from $v_{\text{opt}} = 36$ to $16 \text{ mm}^2/\text{s}$
- with hydraulic fluid based on mineral oils

2) For a speed increase up to $n_{\text{max perm}}$, please observe the diagram on page 7.

3) For further information on HF hydraulic fluids, please see data sheets 90223 and 90225

Permissible radial and axial loading of the drive shaft

Size	NG		18	28	45	71	88	100
Maximum radial force at a/2	 $F_{q \max}$	N	350	1200	1500	1900	1900	2300
Maximum axial force	 $\pm F_{ax \max}$	N	700	1000	1500	2400	2400	4000

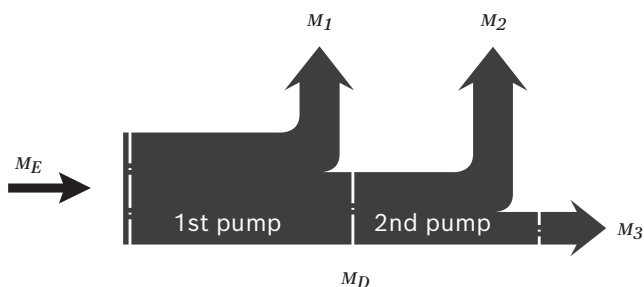
Notice

- The values given are maximum values and do not apply to continuous operation. All loads of the drive shaft reduce the bearing service life!
- For drives with radial loading (pinion, V-belt), please contact us

Permissible inlet and through-drive torques

Size			18	28	45	71	88	100
Torque at $V_{g \max}$ and $\Delta p = 280 \text{ bar}^{1)}$	$M_{\max}$	Nm	80	125	200	316	392	445
Max. input torque on drive shaft ²⁾								
S	$M_{E \max}$	Nm	124	198	319	626	626	1104
	$\varnothing$	in	3/4	7/8	1	1 1/4	1 1/4	1 1/2
R	$M_{E \max}$	Nm	160	250	400	644	644	–
	$\varnothing$	in	3/4	7/8	1	1 1/4	1 1/4	–
P	$M_{E \max}$	Nm	88	137	200	439	439	857
	$\varnothing$	in	18	22	25	32	32	40
Maximum through-drive torque								
S	$M_{D \max}$	Nm	108	160	319	492	492	778
R	$M_{D \max}$	Nm	120	176	365	548	548	–
P	$M_{D \max}$	Nm	88	137	200	439	439	778

▼ Distribution of torques



Torque at 1st pump	M_1
Torque at 2nd pump	M_2
Torque at 3rd pump	M_3
Input torque	$M_E = M_1 + M_2 + M_3$
	$M_E < M_{E \max}$
Through-drive torque	$M_D = M_2 + M_3$
	$M_D < M_{D \max}$

1) Efficiency not considered

2) For drive shafts with no radial force

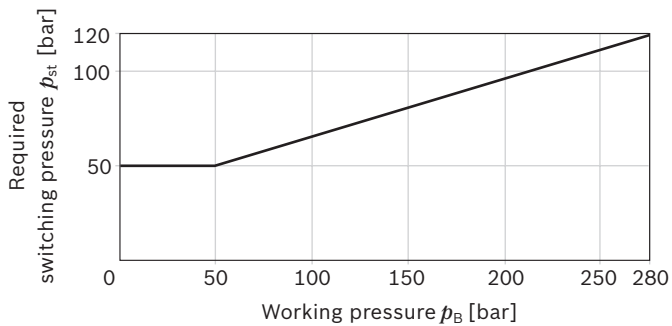
DG – Two-point control, direct operated

The variable pump can be set to a minimum swivel angle by connecting an external switching pressure to port **X**. This will supply control fluid directly to the stroking piston; a minimum control pressure of $p_{st} \geq 50$ bar is required. The variable pump can only be switched between $V_{g\ max}$ and $V_{g\ min}$.

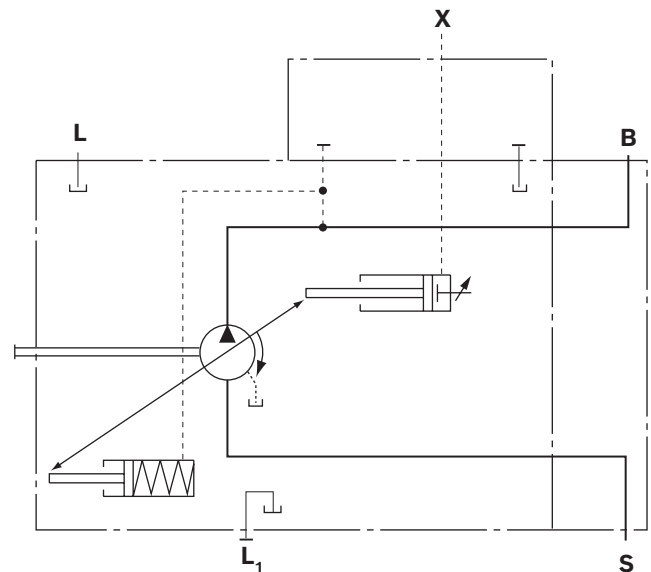
Please note that the required switching pressure at port **X** is directly dependent on the actual working pressure p_B in port **B**. (see switching pressure characteristic curve). The maximum permissible switching pressure is 280 bar.

- Switching pressure p_{st} in **X** = 0 bar $\triangleq V_{g\ max}$
- Switching pressure p_{st} in **X** ≥ 50 bar $\triangleq V_{g\ min}$

▼ Switching pressure characteristic curve



▼ Circuit diagram DG

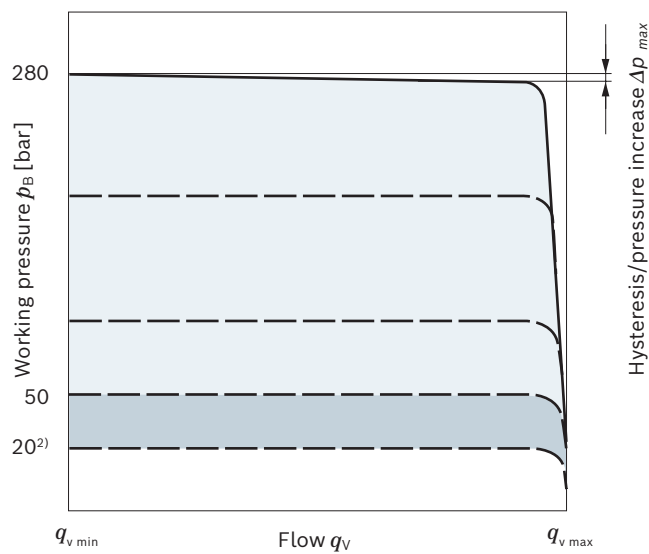


DR – Pressure controller

The pressure controller limits the maximum pressure at the pump outlet within the control range of the variable pump. The variable pump only supplies as much hydraulic fluid as is required by the consumers. If the working pressure exceeds the pressure command value at the pressure valve, the pump will regulate to a smaller displacement to reduce the control differential.

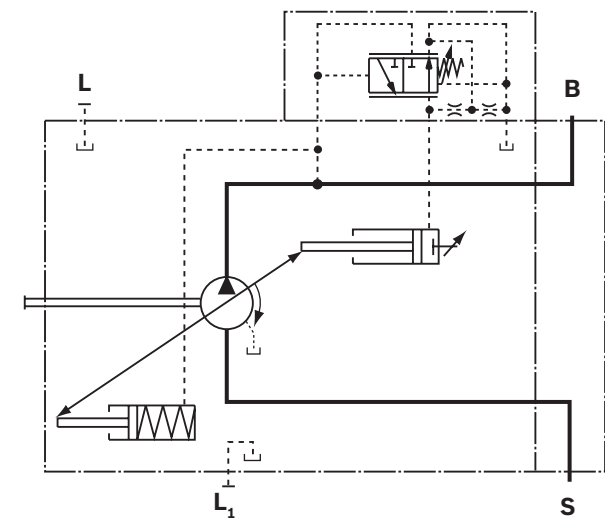
- ▶ Basic position in depressurized state: $V_{g \max}$.
- ▶ Setting range¹⁾ for pressure control 50 to 280 bar. Standard is 280 bar.

Characteristic curve



Characteristic curve valid at $n_1 = 1500 \text{ rpm}$ and $\vartheta_{\text{fluid}} = 50 \text{ }^\circ\text{C}$.

Circuit diagram DR



Controller data DR

NG		18	28	45	71	88	100
Pressure increase	Δp [bar]	4	4	6	8	9	10
Hysteresis and repeatability	Δp [bar]	maximum 3					
Pilot fluid consumption	[l/min]	maximum approx. 3					

1) In order to prevent damage to the pump and the system, the permissible setting range must not be exceeded. The range of possible settings at the valve is higher.

2) For settings below 50 bar, please use the SO275 special pressure controller (setting range: 20 to 100 bar).

DRG – Pressure controller, remotely controlled

For the remote controlled pressure controller, the pressure limitation is performed using a separately arranged pressure relief valve. Therefore, any pressure control value under the pressure set on the pressure controller can be regulated. Pressure controller DR see page 12.

A pressure relief valve is externally piped up to port **X** for remote control. This relief valve is not included in the scope of delivery of the DRG control.

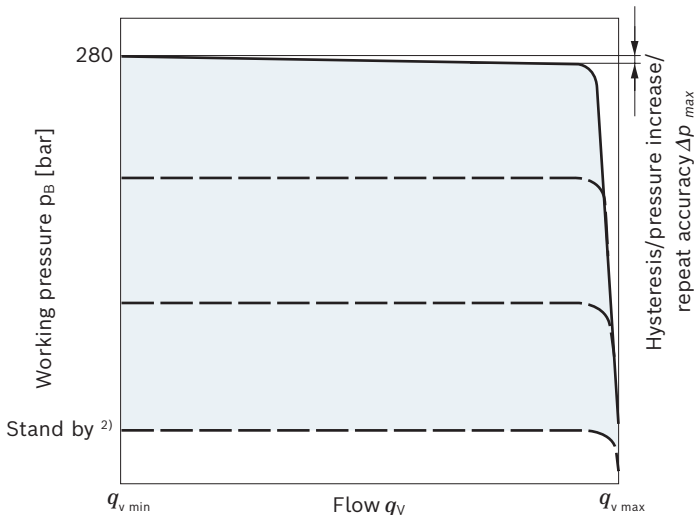
A differential pressure of 20 bar Δp (standard setting) results in a pilot oil flow of approx. 1.5 l/min at port **X**. If another setting is required (range from 10-22 bar) please state in plain text.

As a separate pressure relief valve **(1)** we recommend:

- ▶ A direct operated, hydraulic or electric proportional one, suitable for the quantity of pilot fluid mentioned above. The maximum line length should not exceed 2 m.
- ▶ Basic position in depressurized state: $V_{g \max}$.
- ▶ Setting range¹⁾ for pressure control 50 to 280 bar **(3)**. Standard is 280 bar.
- ▶ Setting range for differential pressure 10 - 22 bar **(2)**. Standard is 20 bar.

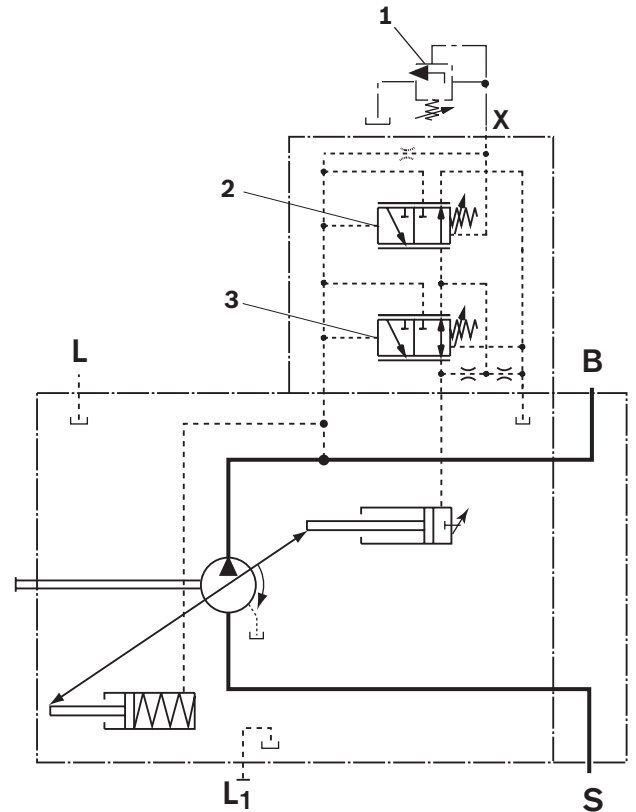
Unloading port **X** to the reservoir results in a zero stroke pressure (standby) which is approx. 1 to 2 bar higher than the defined differential pressure Δp , however system influences are not taken into account.

▼ Characteristic curve of the DRG



Characteristic curve valid at $n_1 = 1500 \text{ rpm}$ and $\theta_{\text{fluid}} = 50^\circ \text{C}$.

▼ Circuit diagram of the DRG



- 1 The separate pressure relief valve and the line are not included in the scope of delivery.
- 2 Remote controlled pressure cut-off **(G)**
- 3 Pressure controller **(DR)**

Controller data DRG

NG		18	28	45	71	88	100
Pressure increase	Δp [bar]	4	4	6	8	9	10
Hysteresis and repeatability	Δp [bar]	maximum 4					
Pilot fluid consumption DR and DRG	[l/min]	maximum approx. 4.5					

1) In order to prevent damage to the pump and the system, the permissible setting range must not be exceeded. The range of possible settings at the valve is higher.

2) Zero stroke pressure from pressure setting Δp on controller **(2)**

DFR/DFR1 – Pressure flow controller

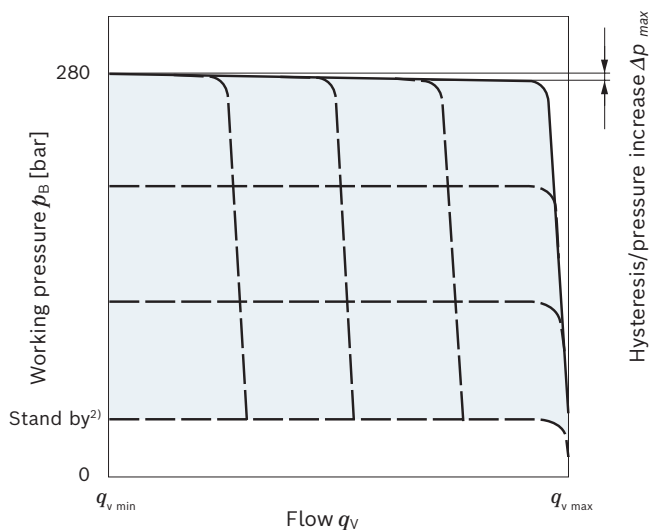
In addition to the pressure controller function (see page 12), an adjustable orifice (e.g. directional valve) is used to adjust the differential pressure upstream and downstream of the orifice. This is used to control the pump flow. The pump flow is equal to the actual hydraulic fluid quantity required by the consumer. With all controller combinations, the V_g reduction has priority.

- ▶ Basic position in depressurized state: $V_{g \max}$.
- ▶ Setting range¹⁾ to 280
Standard is 280 bar.
- ▶ For pressure controller data, see page 12

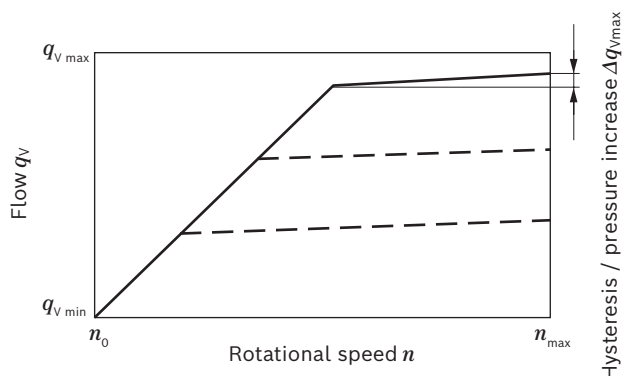
Notice

- ▶ The DFR1 version has no unloading between **X** and the reservoir. The LS must thus be unloaded in the system. Because of the flushing function of the flow controller in the DFR1 control valve, sufficient unloading of the **X**-line must also be provided.

▼ Characteristic curve

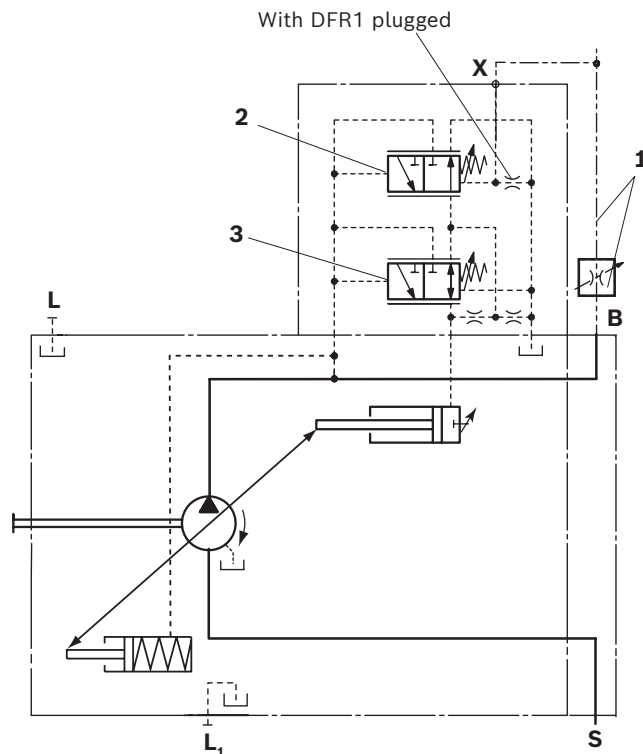


▼ Characteristic curve at variable rotational speed



Characteristic curves valid at $n_1 = 1500$ rpm and $\theta_{fluid} = 50$ °C.

▼ Circuit diagram DFR



- 1 The metering orifice (control block) and the line is not included in the scope of delivery.
- 2 Flow controller (**FR**).
- 3 Pressure controller (**DR**)

For further information see page 15

1) In order to prevent damage to the pump and the system, the permissible setting range must not be exceeded. The range of possible settings at the valve is higher.

2) Zero stroke pressure from pressure setting Δp on controller (2)

Differential pressure Δp :

- Standard setting: 14 bar
 If another setting is required, please state in the plain text.
- Setting range: 14 bar to 22 bar

Relieving the load on port **X** to the reservoir results in a zero stroke pressure ("standby") pressure which lies about 1 to 2 bar higher than the defined differential pressure Δp , however, system influences are not taken into account.

Controller data

DR pressure controller data see page 12

Maximum flow deviation measured at drive speed

$n = 1500$ rpm.

NG		18	28	45	71	88	100
Flow deviation	Δq_{Vmax} [l/min]	0.9	1.0	1.8	2.8	3.4	4.0
Hysteresis and repeatability	Δp [bar]	maximum 4					
Pilot fluid consumption	[l/min]	maximum approx. 3 to 4.5 (DFR) maximum approx. 3 (DFR1)					

ED – Electro-hydraulic pressure control

The ED valve is set to a certain pressure by a specified variable solenoid current.

When changing the consumer (load pressure), this causes an increase or decrease in the pump swivel angle (flow) in order to maintain the electrically set pressure level.

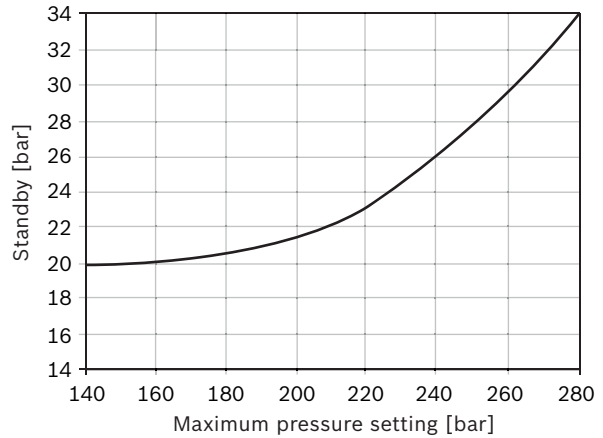
The pump thus only delivers as much hydraulic fluid as the actuators can take. The desired pressure level can be set steplessly by varying the solenoid current.

As the solenoid current signal drops towards zero, the pressure will be limited to $p_{\max}$ by an adjustable hydraulic pressure cut-off (secure fail safe function in case of power failure, e.g. for fan speed control). The swivel time characteristic of the ED control was optimized for the use as a fan drive system.

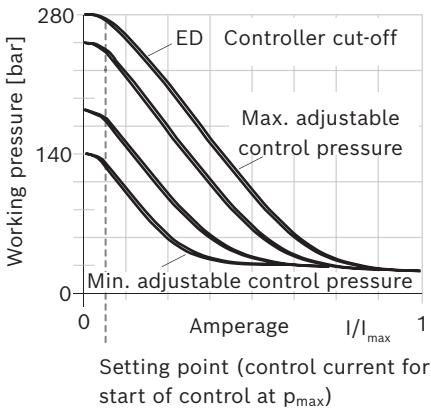
When ordering, specify the type of application in plain text.

- ▶ Pilot fluid consumption: 3 to 4.5 l/min.
- ▶ For standby standard setting, see the following diagram; other values on request.

▼ Influence of the pressure setting on standby (maximally energized)

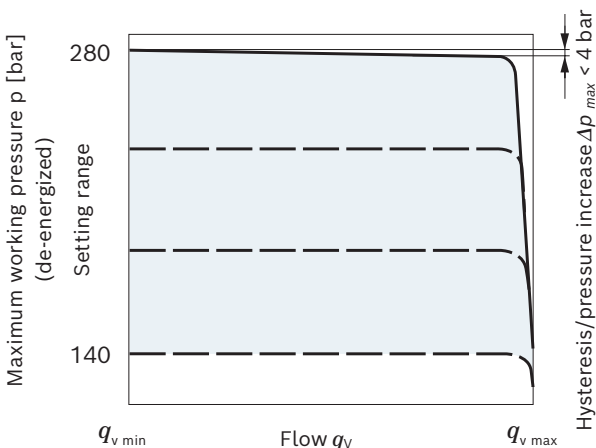


▼ Current/pressure characteristic curve ED (negative characteristic curve)



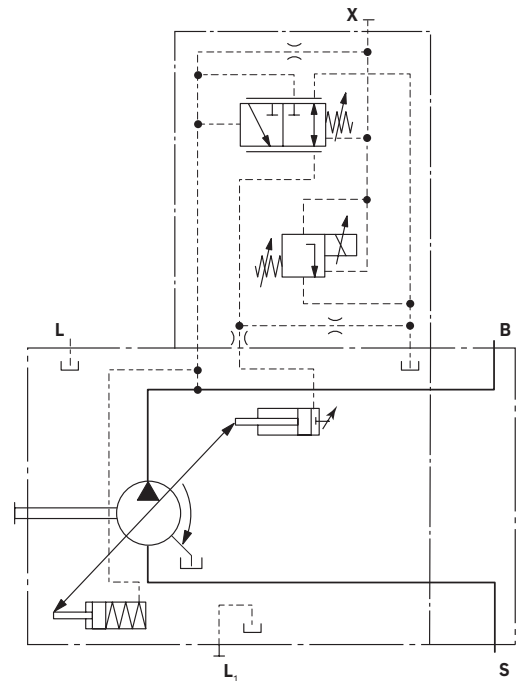
- ▶ Hysteresis static current-pressure characteristic curve < 3 bar.

▼ Flow-pressure characteristic curve



Characteristic curves valid at $n_1 = 1500$ rpm and $\vartheta_{\text{fluid}} = 50$ °C.

▼ Circuit diagram ED72



Technical data, solenoids	ED72
Voltage	24 V ($\pm 20\%$)
Control current	
Start of control at $p_{\max}$	50 mA
Start of control at $p_{\min}$	600 mA
Current limit	0.77 A
Nominal resistance (at 20 °C)	22.7 Ω
Dither frequency	100 Hz
Recommended amplitude peak to peak	120 mA
Duty cycle	100%
Type of protection and control electronics see connector version page 43	
Operating temperature range at valve	-20 °C to +115 °C

Notice!

With **ED72**, de-energized operating condition (jump from 50 to 0 mA) results in a pressure increase of the maximum pressure of 4 to 5 bar.

ER – Electro-hydraulic pressure control

The ER valve is set to a certain pressure by a specified variable solenoid current.

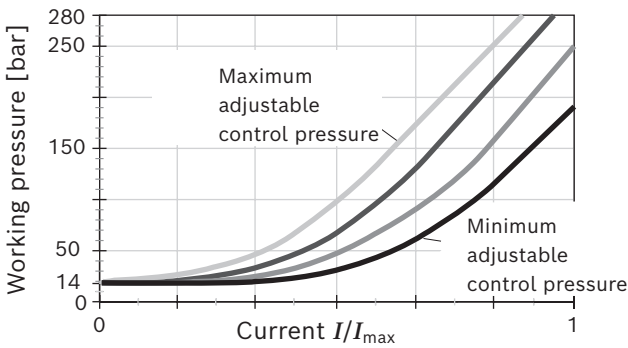
When changing the consumer (load pressure), this causes an increase or decrease in the pump swivel angle (flow) in order to maintain the electrically set pressure level.

The pump thus only delivers as much hydraulic fluid as the consumers can take. The desired pressure level can be set steplessly by varying the solenoid current.

If the solenoid current drops towards zero, the pressure will be limited to $p_{\min}$ (standby) by an adjustable hydraulic pressure cut-off.

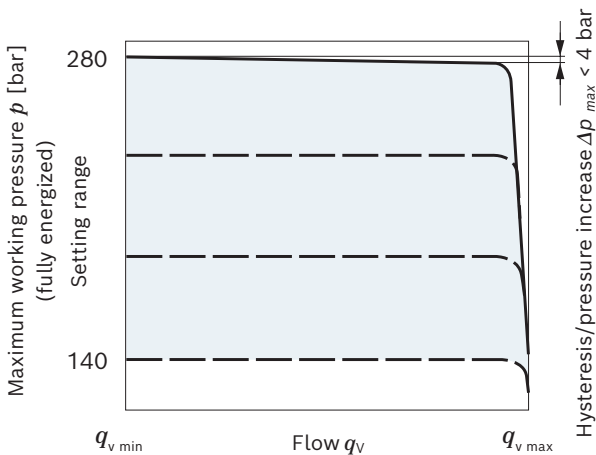
Observe project planning note.

▼ Current-pressure characteristic curve (positive characteristic curve)



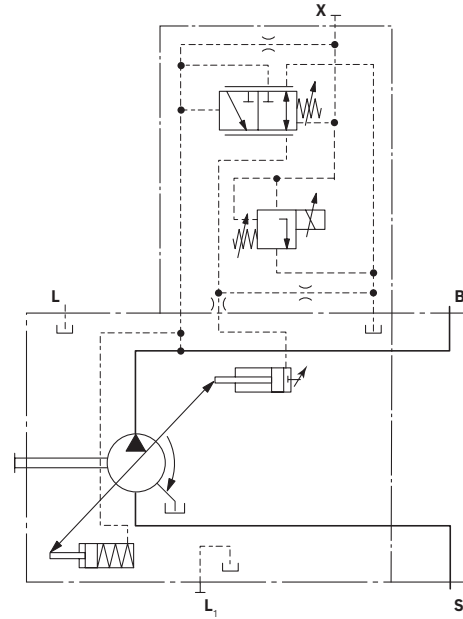
► Hysteresis static < 3 bar.

▼ Flow-pressure characteristic curve



- Characteristic curves valid at $n_1 = 1500$ rpm and $\vartheta_{\text{fluid}} = 50$ °C.
- Pilot fluid consumption: 3 to 4.5 l/min.
- Standby standard setting 14 bar. Other values on request.
- Influence of pressure setting on stand by ± 2 bar

▼ Circuit diagram ER72



Technical data, solenoids	ER72
Voltage	24 V ($\pm 20\%$)
Control current	
Start of control at $p_{\min}$	50 mA
End of control at $p_{\max}$	600 mA
Current limit	0.77 A
Nominal resistance (at 20 °C)	22.7 Ω
Dither frequency	100 Hz
Recommended amplitude peak to peak	120 mA
Duty cycle	100%
Type of protection and control electronics	see connector version page 43
Operating temperature range at valve	-20 °C to +115 °C

Project planning note!

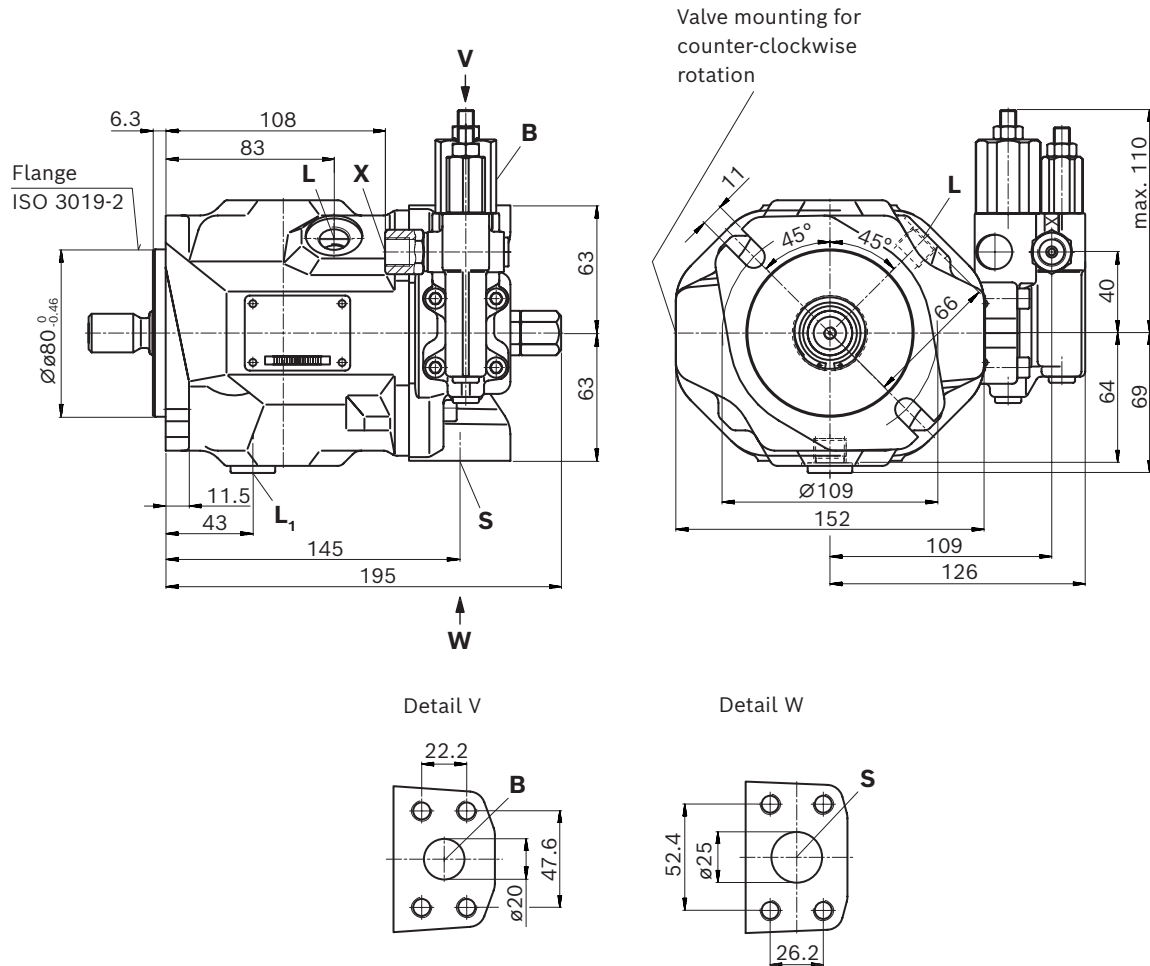
Over-current ($I > 600$ mA at 24 V) to the ER solenoid can result in pressure increases leading to pump or system damage. Therefore:

- Use $I_{\max}$ current limiter solenoids.
- An intermediate plate pressure controller can be used to protect the pump in the event of overflow.

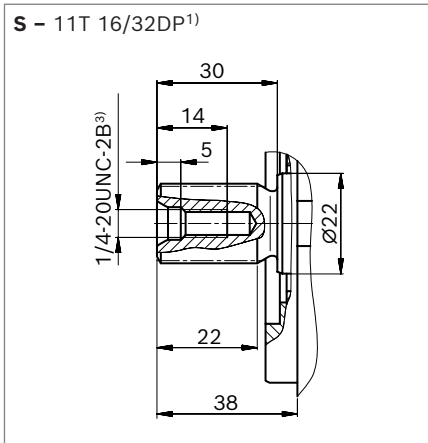
An accessory kit with intermediate plate pressure controller can be ordered from Bosch Rexroth under part number R902490825.

Dimensions, size 18

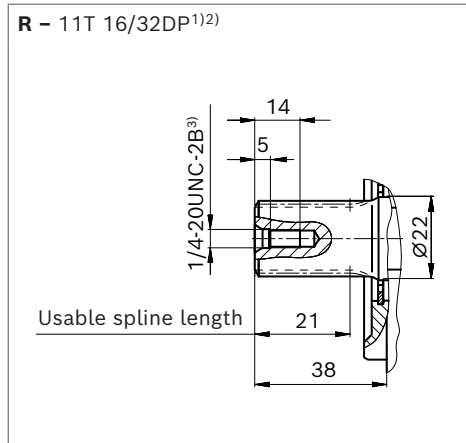
DFR/DFR1 – Pressure flow controller, hydraulic, clockwise rotation



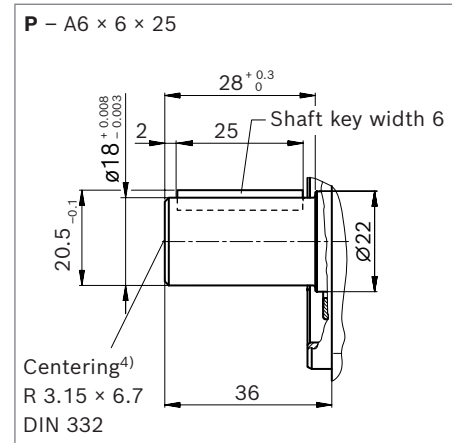
▼ Splined shaft 3/4 in (19-4, ISO 3019-1)



▼ **Splined shaft 3/4 in (similar to ISO 3019-1)**



▼ **Parallel keyed shaft, DIN 6885**

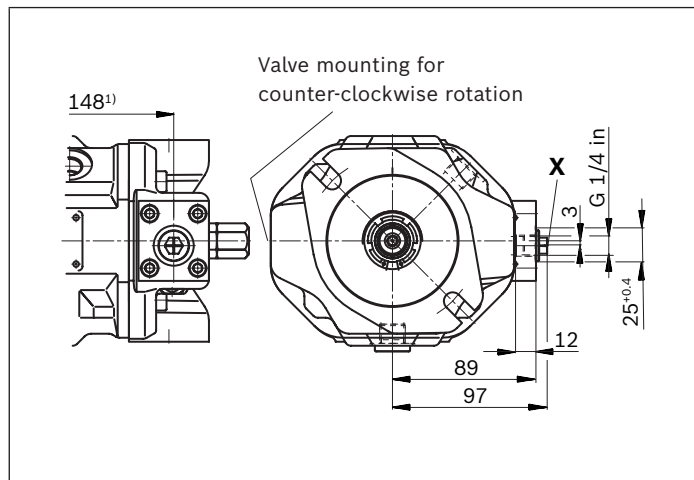


Ports		Standard	Size	$p_{\max}$ [bar] ⁵⁾	State ⁸⁾
B	Working port (standard pressure series) Fastening thread	ISO 6162-1 DIN 13	3/4 in M10 × 1.5; 17 deep	350	O
S	Suction port (standard pressure series) Fastening thread	ISO 6162-1 DIN 13	1 in M10 × 1.5; 17 deep	10	O
L	Drain port	DIN 3852 ⁶⁾	M16 × 1.5; 12 deep	2	O ⁷⁾
L₁	Drain port	DIN 3852 ⁶⁾	M16 × 1.5; 12 deep	2	X ⁷⁾
X	Pilot pressure port	DIN 3852	M14 × 1.5; 12 deep	350	O
X	Pilot pressure port with DG-control	DIN 3852-2	G1/4 in; 12 deep	350	O

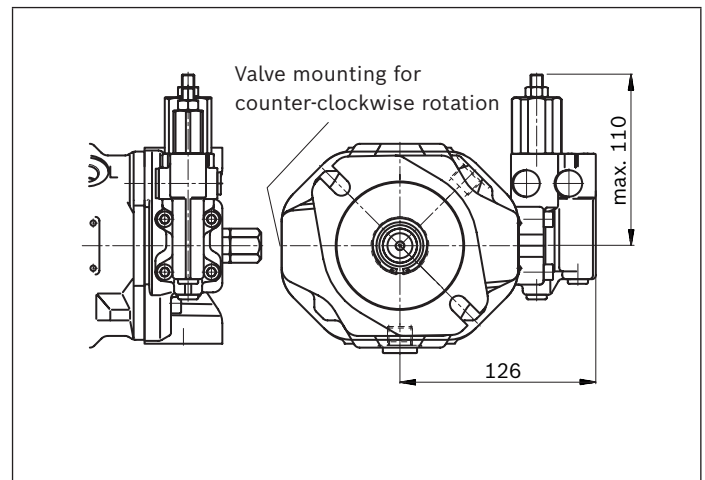
- 1) Involute spline according to ANSI B92.1a, 30° pressure angle, flat root, side fit, tolerance class 5
- 2) Splines according to ANSI B92.1a, spline runout is a deviation from standard ISO 3019-1.
- 3) Thread according to ASME B1.1
- 4) Coupling axially secured, e.g. with a clamp coupling or radially mounted clamping screw

- 5) Depending on the application, momentary pressure peaks can occur. Keep this in mind when selecting measuring devices and fittings.
- 6) The countersink may be deeper than specified in the standard.
- 7) Depending on the installation position, L or L₁ must be connected (also see installation instructions on page 44).
- 8) O = Must be connected (plugged on delivery)
X = Plugged (in normal operation)

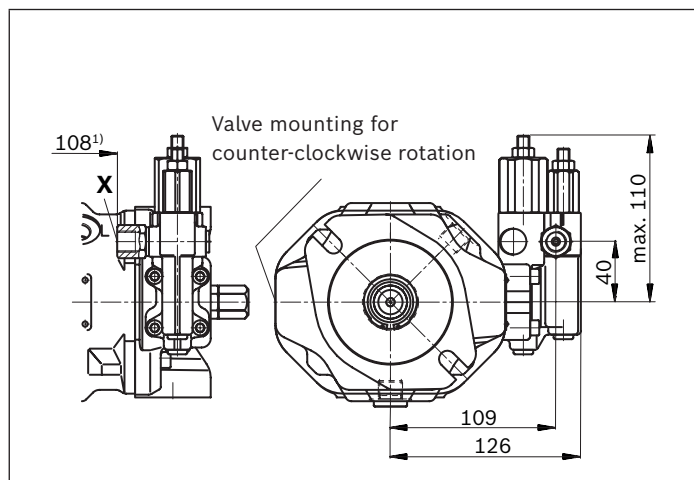
▼ **DG – Two-point control, direct operated**



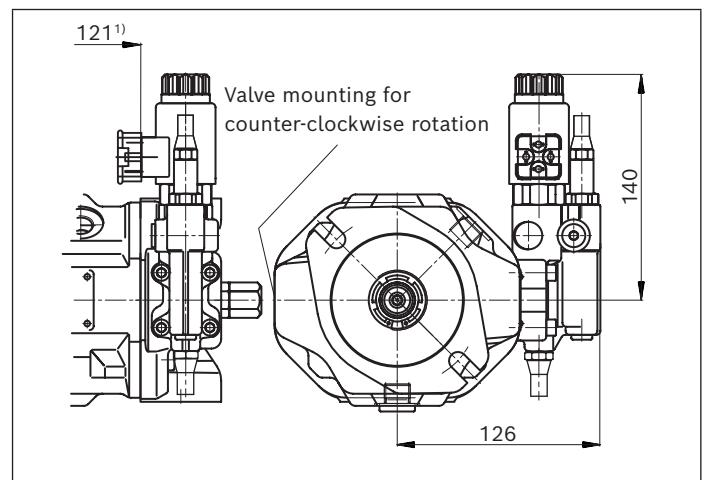
▼ **DR – Pressure controller**



▼ **DRG – Pressure controller, remotely controlled**



▼ **ED7., ER7. – Electro-hydraulic pressure control**

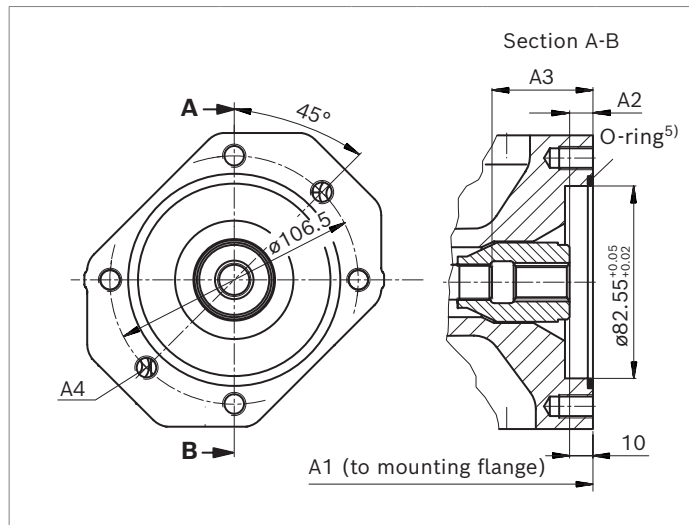


1) To flange surface

Dimensions, through drive**For flanges and shafts according to ISO 3019-1**

Flange (SAE)		Hub for splined shaft ¹⁾		Availability across sizes						Code
Diameter	Mounting ⁴⁾	Diameter		18	28	45	71	88	100	
82-2 (A)	8, 8P, ∞	5/8 in	9T 16/32DP	•	•	•	•	•	•	K01
		3/4 in	11T 16/32DP	•	•	•	•	•	•	K52

• = Available - = Not available

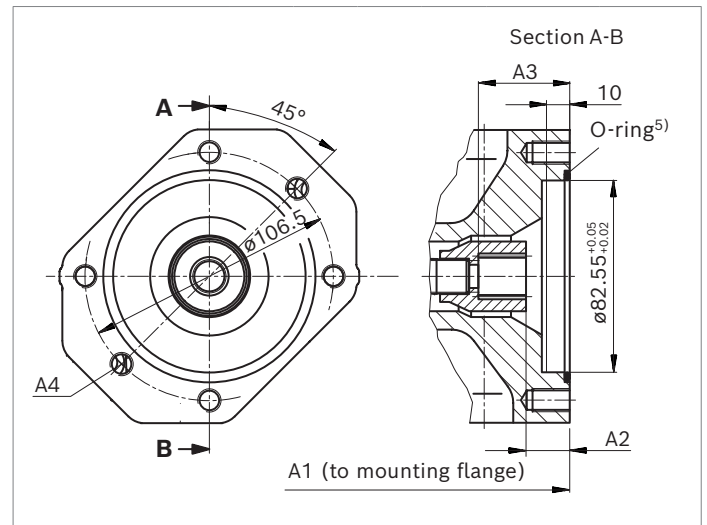
▼ **82-2**

K01 (16-4 (A))	NG	A1	A2 ³⁾	A3 ³⁾	A4 ²⁾
	18	182	9.3	42.5	M10×1.5; 14.5 deep
	28	204	9.2	36.2	M10×1.5; 16 deep
	45	229	10.1	52.7	M10×1.5; 16 deep
	71	267	11.2	60.6	M10×1.5; 20 deep
	88	267	11.2	60.6	M10×1.5; 20 deep
	100	338	10.0	64.3	M10×1.5; 16 deep

1) According to ANSI B92.1a, 30° pressure angle, flat root, side fit, tolerance class 5

2) Thread according to DIN 13

3) Minimum dimensions

▼ **82-2**

K52 (19-4 (A-B))	NG	A1	A2 ³⁾	A3 ³⁾	A4 ²⁾
	18	182	18.3	39.2	M10×1.5; 14.5 deep
	28	204	18.4	39.4	M10×1.5; 16 deep
	45	229	18.4	38.8	M10×1.5; 16 deep
	71	267	20.8	41.2	M10×1.5; 20 deep
	88	267	20.8	41.2	M10×1.5; 20 deep
	100	338	18.6	39.6	M10×1.5; 16 deep

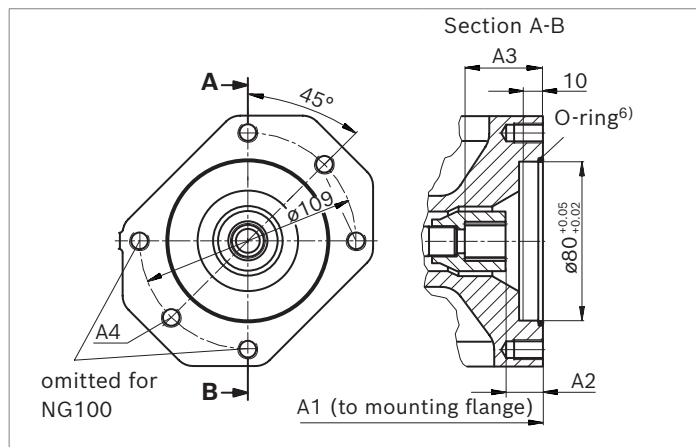
4) Mounting holes pattern viewed on through drive with control at top

5) O-ring included in the scope of delivery

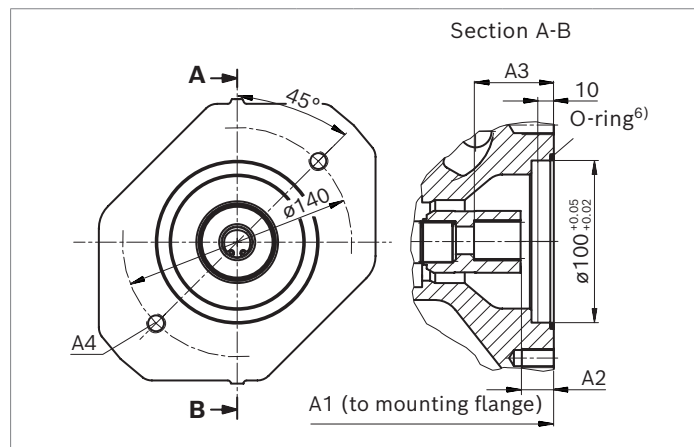
For flanges according to ISO 3019-2 and shafts according to ISO 3019-1

Flange ISO 3019-2		Hub for splined shaft ¹⁾		Availability across sizes						Code
Diameter	Mounting ⁵⁾	Diameter		18	28	45	71	88	100	
80, 2-hole	⌀, o-o, o°	3/4 in	11T 16/32DP	●	●	●	●	●	●	KB2
100, 2-hole	o°	7/8 in	13T 16/32DP	-	●	●	●	●	●	KB3
		1 in	15T 16/32DP	-	-	●	●	●	●	KB4

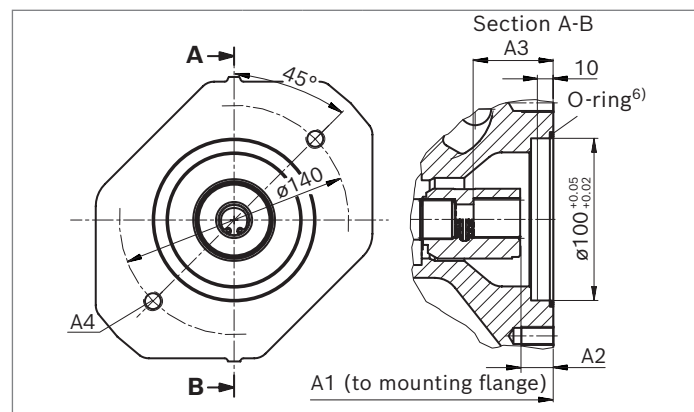
● = Available - = Not available

▼ 80, 2-hole

KB2	NG	A1	A2 ⁴⁾	A3 ⁴⁾	A4 ²⁾
(ISO 3019-1 19-4 (A-B))					
18	182	18.3	39.2	M10×1.5; 14.5 deep	
28	204	18.4	39.4	M10×1.5; 16 deep	
45	229	18.4	38.8	M10×1.5; 16 deep	
71	267	20.8	41.2	M10×1.5; 20 deep	
88	267	20.8	41.2	M10×1.5; 20 deep	
100	338	18.6	39.6	M10×1.5; 20 deep	

▼ 100, 2-hole

KB3	NG	A1	A2 ⁴⁾	A3 ⁴⁾	A4 ²⁾
(ISO 3019-1 22-4 (B))					
28	204	17.4	42.4	M12×1.5 ³⁾	
45	229	17.4	41.8	M12×1.5 ³⁾	
71	267	19.8	44.2	M12×1.5; 20 deep	
88	267	19.8	44.2	M12×1.5; 20 deep	
100	338	17.6	41.9	M12×1.5; 20 deep	

▼ 100, 2-hole

KB4	NG	A1	A2	A3	A4 ²⁾
(ISO 3019-1 25-4 (B-B))					
45	229	17.9	47.4	M12×1.75 ³⁾	
71	267	20.3	49.2	M12 × 1.75; 20 deep	
88	267	20.3	49.2	M12 × 1.75; 20 deep	
100	338	17.8	46.6	M12 × 1.75; 20 deep	

1) According to ANSI B92.1a, 30° pressure angle, flat root, side fit, tolerance class 5

2) Thread according to DIN 13

3) Continuous

4) Minimum dimension

5) Mounting holes pattern viewed on through drive with control at top

6) O-ring included in the scope of delivery

Overview of mounting options

SAE – mounting flange

Through drive			Mounting options – 2nd pump			
Flange ISO 3019-1	Hub for splined shaft	Code	A10VSO/31 NG (shaft)	A10V(S)O/5x NG (shaft)	Gear/gerotor/vane pump	Through drive available for NG
82-2 (A)	5/8 in	K01	–	10 (U) 18 (U)	AZPF, PGH2, PGH3	18 to 100
	3/4 in	K52	–	10 (S) 18 (S, R)	–	18 to 100
101-2 (B)	7/8 in	K68	–	28 (S, R) 45 (U, W) ¹⁾	AZPN, AZPG	28 to 100
	1 in	K04	–	45 (S, R) 60, 63, 72 (U, W) ²⁾	PGH4	45 to 100
127-2 (C)	1 1/4 in	K07	–	60, 63 (S, R) 85 (U) ³⁾ 100 (U) ³⁾	PVV BG 4, 5	71 to 100
	1 1/2 in	K24	–	85 (S) 100 (S)	PGH5	100

ISO – mounting flange

Through drive			Mounting options – 2nd pump			
Flange ISO 3019-2	Hub for splined shaft	Code	A10VSO/31 NG (shaft)	A10V(S)O/5x NG (shaft)	Gerotor pump	Through drive available for NG
80, 2-hole	3/4 in	KB2	18 (S, R)	10 (S)	PGZ	18 to 100
100, 2-hole	7/8 in	KB3	28 (S, R)	–	PGZ	28 to 100
	1 in	KB4	45 (S, R)	–	–	45 to 100
125, 2-hole	1 1/4 in	KB5	71 (S, R) 88 (S, R)	–	–	71 to 100
	1 1/2 in	KB6	100 (S)	–	–	100

ISO – mounting flange for keyed shaft

Through drive			Mounting options – 2nd pump			
Flange similar to ISO 3019-2	Hub for keyed shaft	Code			Radial piston pump	Through drive available for NG
63, 4-hole metric	3/4 in	K57			R4	28 to 100

¹⁾ Not for main pump NG28 with K68
²⁾ Not for main pump NG45 with K04
³⁾ Not for main pump NG71 and NG88 with K07

Combination pumps A10VSO + A10VSO

By using combination pumps, it is possible to have independent circuits without the need for splitter gearboxes. When ordering combination pumps the type designations for the 1st and the 2nd pump must be joined by a "+".

Order example:

A10VSO100DFR1/31R-VSA12KB4+

A10VSO45DFR/31R-VSA12N00

If no further pumps are to be mounted at the factory, the simple type designation is sufficient.

A tandem pump, with two pumps of equal size, is permissible without additional supports, assuming that the dynamic mass acceleration does not exceed maximum 10 g (= 98.1 m/s²).

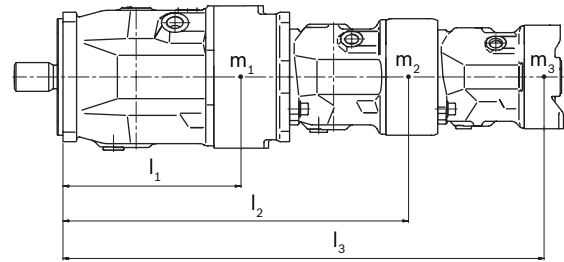
For combination pumps consisting of more than two pumps, a calculation of the mounting flange regarding the permissible mass torque is required (please contact us).

Through drives are plugged with a **non-pressure-resistant** cover. Therefore, single pumps must be equipped with a pressure-resistant cover before commissioning. Through drives can also be ordered with a pressure-resistant cover, please specify in plain text.

Notice

Through drives with installed hub are supplied with a spacer.

The spacer must be removed before installation of the 2nd pump and before commissioning. For information, please refer to the 92711-01-B operating instructions.



m_1, m_2, m_3	Weight of pump	[kg]
l_1, l_2, l_3	Distance from center of gravity	[mm]
$T_m = (m_1 \times l_1 + m_2 \times l_2 + m_3 \times l_3) \times \frac{1}{102} \text{ [Nm]}$		

Calculation for multiple pumps

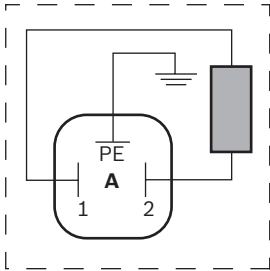
l_1	= Front pump distance from center of gravity (values from "Permissible moments of inertia" table)
l_2	= Dimension "A1" from through drive drawings (page 35 to 40) + l_1 of the 2nd pump
l_3	= Dimension "A1" from through drive drawings (page 35 to 40) of the 1st pump + "A1" of the 2nd pump + l_1 of the 3rd pump

Permissible moments of inertia

Size			18	28	45	71	88	100
static	T_m	Nm	500	880	1370	2160	2160	3000
dynamic at 10 g (98.1 m/s ²)	T_m	Nm	50	88	137	216	216	300
Weight without through drive (N00)	m	kg	12.9	18	23.5	35.2	35.2	49.5
Weight with through drive (K..)			13.8	19.3	25.1	38	38	55.4
Distance, center of gravity without through drive (N00)	l_1	mm	92	100	113	127	127	161
Distance, center of gravity with through drive (K..)	l_1	mm	98	107	120	137	137	178

Connector for solenoids

Device plug on solenoid (version H)
according to DIN EN 175301-803-A002M



With correctly mounted mating connector, the following type of protection can be achieved:

- IP65 (DIN/EN 60529)

Notice

- If necessary, you can change the position of the connector by turning the solenoid body.
- The procedure is defined in the operating instructions 92711-01-B .

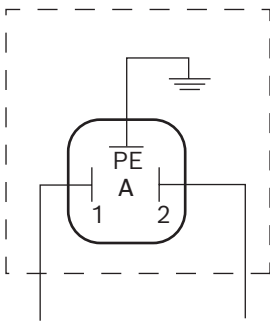
Mating connector

HIRSCHMANN **DIN EN 175301-803-A002F**

without bidirectional suppressor diode **H**

The mating connector (plug-in connector) is not included in the scope of delivery.

This can be supplied by Bosch Rexroth on request, under Bosch Rexroth material number: R902602623



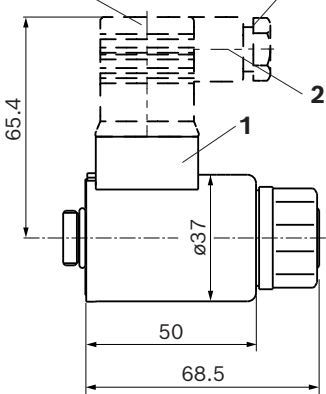
Control electronics

24 V nominal voltage, for ED72/ER72

Control	Electronics function	Electronics	Further information
Electric pressure control	Valve amplifier for proportional valves without electrical position feedback	VT-MSPA1	analog 30232

Mounting bolt M3
tightening torque:
 $M_A = 0.5 \text{ Nm}$

Cable fitting M16x1.5
tightening torque:
 $M_A = 1.5 - 2.5 \text{ Nm}$



- 1 Device plug on the solenoid
- 2 Mating connector (not included in the scope of delivery)

The seal ring in the cable fitting is suitable for lines of diameter 4.5mm to 10mm.

Installation instructions

General

The axial piston unit must be filled with hydraulic fluid and air bled during commissioning and operation. This must also be observed following a longer standstill as the axial piston unit may empty via the hydraulic lines. Particularly with the "drive shaft up/down" installation position, filling and air bleeding must be carried out completely as there is, for example, a danger of dry running. The leakage in the housing area must be directed to the reservoir via the highest available drain port (**L**, **L₁**). For combination pumps, the leakage must be drained off at each single pump.

If a shared drain line is used for several units, make sure that the respective case pressure in each unit is not exceeded. The shared drain line must be dimensioned to ensure that the maximum permissible case pressure of all connected units is not exceeded in any operating condition, particularly at cold start. If this is not possible, separate drain lines must be laid if necessary.

To prevent the transmission of structure-borne noise, use elastic elements to decouple all connecting lines from all vibration-capable components (e.g. reservoir, frame parts). Under all operating conditions, the suction lines and the drain lines must flow into the reservoir below the minimum fluid level. The permissible suction height h_s results from the total pressure loss. However, it must not be higher than $h_{S\ max} = 800\text{ mm}$. The minimum suction pressure at port **S** must not fall below 0.8 bar absolute during operation and during cold start.

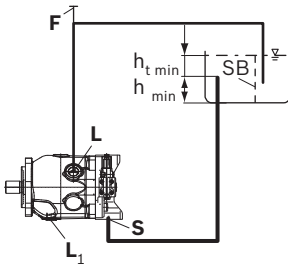
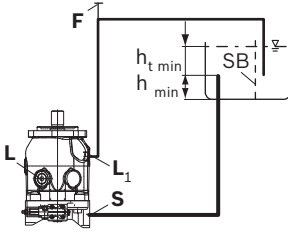
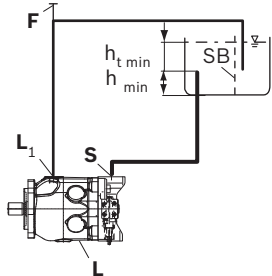
For the reservoir design, ensure that there is an adequate distance between the suction line and the drain line. We recommend using a baffle (baffle plate) between suction line and drain line. A baffle improves the air separation ability as it gives the hydraulic fluid more time for desorption. Apart from that, this prevents the heated return flow from being drawn directly back into the suction line. The suction port must be supplied with air-free, "calmed" and cooled hydraulic fluid. For key, see page 46.

Installation position

See the following examples **1** to **9**. Further installation positions are available upon request. Recommended installation position: **1** and **3**

Below-reservoir installation (standard)

Below-reservoir installation means that the axial piston unit is installed outside of the reservoir below the minimum fluid level.

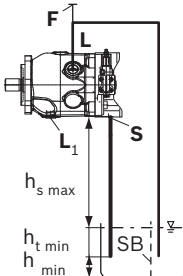
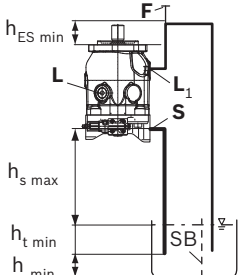
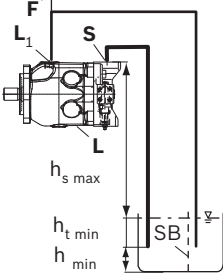
Installation position	Air bleed	Filling
1	F	L (F)
		
2 ¹⁾	F	L ₁ (F)
		
3	F	L ₁ (F)
		

1) Because complete air bleeding and filling are not possible in this position, the pump should be air bled and filled in a horizontal position before installation.

Above-reservoir installation

Above-reservoir installation means that the axial piston unit is installed above the minimum fluid level of the reservoir. To prevent the axial piston unit from draining in position 5, the height difference $h_{ES\ min}$ must be at least 25 mm. Observe the maximum permissible suction height $h_{S\ max} = 800\ mm$

A check valve in the drain line is only permissible in individual cases. Consult us for approval.

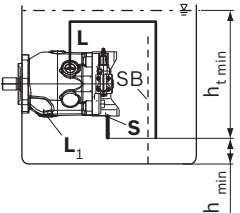
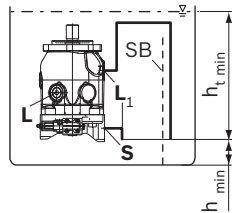
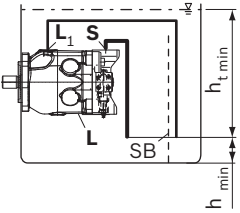
Installation position	Air bleed	Filling
4 	F	L (F)
5¹⁾ 	F	L₁ (F)
6 	F	L₁ (F)

For key, see page 46.

Inside-reservoir installation

Inside-reservoir installation is when the axial piston unit is installed in the reservoir below the minimum fluid level. The axial piston unit is completely below the hydraulic fluid. If the minimum fluid level is equal to or below the upper edge of the pump, see chapter "Above-reservoir installation".

Axial piston units with electrical components (e.g. electric control, sensors) may not be installed in a reservoir below the fluid level.

Installation position	Air bleed	Filling
7 	Via the highest available port L	Automatically via the open port L or L₁ due to the position under the hydraulic fluid level
8¹⁾ 	Via the highest available port L₁	Automatically via the open port L , L₁ due to the position under the hydraulic fluid level
9 	Via the highest available port L₁	Automatically via the open port L or L₁ due to the position under the hydraulic fluid level

¹⁾ Because complete air bleeding and filling are not possible in this position, the pump should be air bled and filled in a horizontal position before installation.

Key	
F	Filling / Air bleeding
S	Suction port
L; L ₁	Drain port
SB	Baffle (baffle plate)
h _{t min}	Minimum required immersion depth (200 mm)
h _{min}	Minimum required distance to reservoir bottom (100 mm)
h _{ES min}	Minimum height required to prevent axial piston unit from draining (25 mm)
h _{S max}	Maximum permissible suction height (800 mm)

Notice

Port **F** is part of the external piping and must be provided on the customer side to simplify the filling and air bleeding.

Project planning notes

- ▶ The A10VSO axial piston variable pump is intended to be used in open circuit.
- ▶ Project planning, installation and commissioning of the axial piston units requires the involvement of skilled personnel.
- ▶ Before using the axial piston unit, please read the corresponding instruction manual completely and thoroughly. If necessary, this can be requested from Bosch Rexroth.
- ▶ Before finalizing your design, request a binding installation drawing.
- ▶ The specified data and notes contained herein must be observed.
- ▶ Depending on the operating conditions of the axial piston unit (working pressure, fluid temperature), the characteristic curve may shift.
- ▶ The characteristic curve may also shift due to the dither frequency or control electronics.
- ▶ Preservation: Our axial piston units are supplied as standard with preservation protection for a maximum of 12 months. If longer preservation protection is required (maximum 24 months), please specify this in plain text when placing your order. The preservation periods apply under optimal storage conditions, details of which can be found in the data sheet 90312 or the instruction manual.
- ▶ Not all versions of the product are approved for use in a safety function according to ISO 13849. Please consult the proper contact at Bosch Rexroth if you require reliability parameters (e.g. MTTF_d) for functional safety.
- ▶ Depending on the type of control used, electromagnetic effects can be produced when using solenoids. Applying a direct voltage signal (DC) to solenoids does not create electromagnetic interference (EMI) nor is the solenoid affected by EMI. Electromagnetic interference (EMI) potential exists when operating and controlling a solenoid with a modulated direct voltage signal (e.g. PWM signal). Appropriate testing and measures should be taken by the machine manufacturer to ensure other components or operators (e.g. with pacemaker) are not affected by this potential.
- ▶ Pressure controllers are not safeguards against pressure overload. Be sure to add a pressure relief valve to the hydraulic system.
- ▶ Please note that a hydraulic system is an oscillating system. This can lead, for example, to the excitation of the natural frequency within the hydraulic system during operation at constant rotational speed over a long period of time. The excitation frequency of the pump is 9 times the rotational speed frequency. This can be prevented, for example, with suitably designed hydraulic lines.
- ▶ Please note the details regarding the tightening torques of port threads and other threaded joints in the instruction manual.
- ▶ The ports and fastening threads are designed for the $p_{\max}$ permissible pressures of the respective ports, see the connection tables. The machine or system manufacturer must ensure that the connecting elements and lines correspond to the specified application conditions (pressure, flow, hydraulic fluid, temperature) with the necessary safety factors.
- ▶ The service ports and function ports are only intended to accommodate hydraulic lines.

Safety instructions

- ▶ During and shortly after operation, there is a risk of getting burnt on the axial piston unit and especially on the solenoids. Take the appropriate safety measures (e.g. by wearing protective clothing).
- ▶ Moving parts in control equipment (e.g. valve spools) can, under certain circumstances, get stuck in position as a result of contamination (e.g. contaminated hydraulic fluid, abrasion, or residual dirt from components). As a result, the hydraulic fluid flow and the build-up of torque in the axial piston unit can no longer respond correctly to the operator's specifications. Even the use of various filter elements (external or internal flow filtration) will not rule out a fault but merely reduce the risk. The machine/system manufacturer should test whether additional measures are required on the machine for the relevant application in order to bring the driven consumer into a safe position (e.g. safe stop) and make sure any measures are properly implemented.